

A Comparative Study on The Rotational Vibration Measurement Using Ordinary Incremental and Absolute Encoders

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ABSTRACT

The present study evaluates two procedures for measuring the rotational vibration of a shaft, using an incremental and an absolute encoder, respectively. First, the procedures for rotational vibration measurement using the two encoders are explained. The validity of these procedures is examined through computer simulations, which demonstrate that both procedures are accurate in measuring rotational vibration. Furthermore, the simulation results for both procedures exhibit good correlations in both the time and frequency domains. Finally, the procedures are experimentally verified using a simple test setup with a universal joint to generate rotational vibration. The test results match well with each other and with the theoretical data, confirming that the proposed method is capable of accurately measuring the rotational vibration of rotating machines.

요 약

일반적인 증분형 인코더와 절대위치 인코더를 이용하여 회전진동을 측정하는 두 방법을 비교하여 검토하였다. 우선 두 인코더를 이용하여 진동을 측정하는 과정을 설명하였다. 다음으로 컴퓨터 시뮬레이션을 이용하여 두 방법의 타당성을 검토하였으며 결과로 두 방법이 회전진동 측정에 충분한 정확도를 가지고 있으며 두 방법의 결과가 잘 일치하고 있음을 확인하였다. 마지막으로 유니버설 조인트로 연결된 회전축의 회전진동을 두 방법을 적용하여 측정하였다. 두 결과는 서로 잘 일치하며 이론적으로 예측된 값과 잘 부합되는 것을 확인하였다. 결론적으로 상기 두 방법은 회전기계의 회전진동을 정확하게 측정할 수 있음을 확인하였다.

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1. Introduction

Rotational or torsional vibration which are commonly generated in mechanical or electric system often deteriorate system performances and decreases system endurance limits given large amplitude⁽¹⁻⁶⁾. So, the measurement of rotational or torsional vibration is very important for the system inspection and maintenance. Various methods to measure torsional and rotational vibration have been studied and implemented. These methods include angular accelerometer method⁽⁷⁾, laser interferometer⁽⁸⁾, video images analysis⁽⁹⁾ and encoder method⁽¹⁰⁻¹²⁾.

Recently encoders are being widely used due to simple application relatively low cost etc. Two types of encoders, incremental encoder and absolute encoder, are being used in the real applications. In the incremental encoder, a pulse will be generated for the specific amount of rotation of the shaft. On the other hand, the absolute encoder gives voltage output proportional to the rotation angle of the shaft.

Application of incremental encoder for the measurement of rotational or torsional vibration has been studied for a long time and several commercial products have been introduced also. Negrea et al. made analytical comparison of the various methods to reduce or compensate the errors in the measurement of rotational vibration using incremental encoders⁽¹¹⁾. Nam et al. proposed a correction process for the rotational vibration measurement system using incremental encoders⁽¹³⁾. The studies on the application of absolute encoders for the measurement of rotational or torsional vibration has not been sufficiently studied thus far. Recently Lee studied the measurement of rotational vibration with ordinary absolute encoders showing that the method has sufficient accuracy in the measurement of rotational vibration⁽¹⁴⁾.

In this study the measurement of rotational vibration with two types of encoders will be comparatively investigated. The measurement processes of rotational displacement and velocity using in-

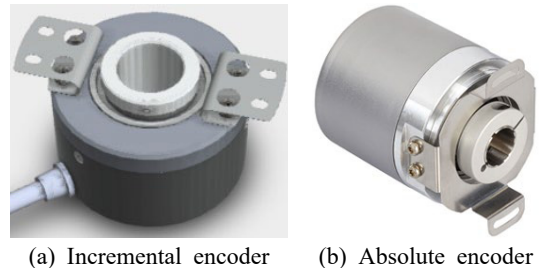


Fig. 1 Examples of the ordinary encoder

dividual encoders are explained in Sections 2 and 3. Also, in Section 4, the numerical validation using computer software and experimental verifications using the simple test setup with a universal joint for the two processes are provided. Finally, the conclusions of this study are given in Section 5.

2. Measuring Shaft Rotation with Encoders

As stated in the previous section, two types of encoders are frequently used for the measurement of the rotational displacement. Fig. 1(a) and Fig. 1(b) shows some examples of ordinary encoders.

Each type of encoder has its own specific mechanism for the measurement of rotational displacement. These mechanisms will be explained in this section and will be used to develop the processes for the rotational vibration measurement of two types of encoders.

2.1 Rotational Displacement Measurement Using an Incremental Encoder

An ordinary incremental encoder that generates a square pulse per pre-determined rotation angle of the shaft is used in this study. As explained above, the incremental encoder generates a specific number (pulses per revolution: PPR) of pulses per revolution. In case of no rotational vibration or disturbance, the output of the encoder would be in the shape of that in Fig. 2.

If the numbers of pulses from the encoder is N , the rotation angle (θ) of the shaft can be calculated with Eq. (1) by dividing $2\pi N$ with PPR.

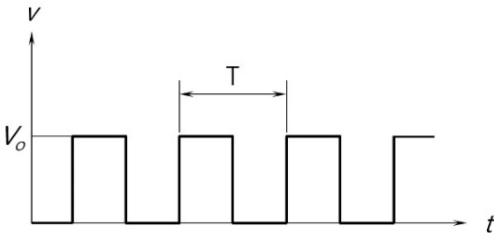


Fig. 2 Example of the output signal from an incremental encoder

$$\theta = 2\pi N / PPR \tag{1}$$

2.2 Rotational Displacement Measurement Using an Absolute Encoder

The absolute encoder generates the voltage output proportional to the rotational angle of the shaft from a specific reference point. An example voltage output from the absolute encoder is explained in Fig. 3 where the output voltage increases linearly from V_{min} to V_{max} according to the rotation angle in the range from 0 to θ_{max} . In this plot the disturbances and rotational vibration of the shaft are assumed to be negligible. And the rotation angle of the shaft at a specific time slot can be obtained by measuring the output voltage at the given instant. Also, the number of rotations of the shaft can be calculated with the number of triangular pulses in the given time interval.

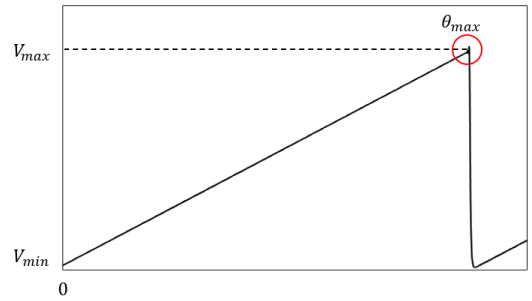


Fig. 3 Example of the output signal from an absolute encoder

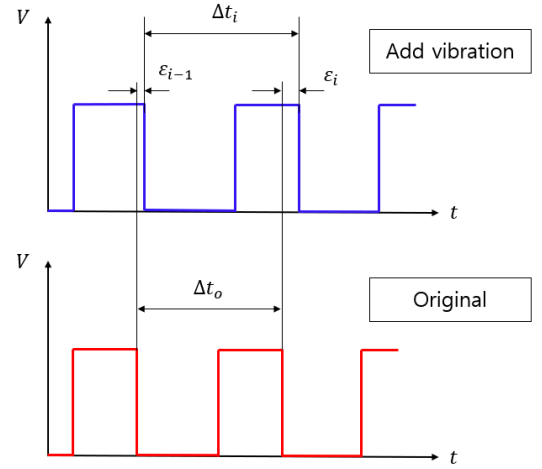


Fig. 4 Pulse output of incremental encoder installed at shaft with and without torsional vibration

3. Measurement of Rotational Vibration Using Encoders

In this section, the processes for the rotational vibration measurement of the two types of encoders based on their rotational displacement measurement mechanisms are explained. The processes are developed based on the characteristics of the output voltage of each type of encoder.

3.1 Rotational Vibration Measurement Using Incremental Encoder

In this section, the process for the rotational vi-

bration measurement of the incremental encoder is provided. In case of the shaft rotating with a constant angular velocity, the incremental encoder will generate pulses in a constant time interval (period) as shown in Fig. 2. But in the case of variable angular velocity due to the rotational vibration, the encoder will generate a series of pulses with variable periods. So, it is possible to find out the variation of angular velocity of the shaft by checking the periods of the individual pulses.

The angular velocity of the shaft corresponding to the i^{th} pulse in Fig. 4 can be calculated as Eq. (2), following equation.

$$\omega_i = \frac{\Delta \theta}{\Delta t_i} = \frac{2\pi / PPR}{\Delta t_i} \text{ (rad/s)} \tag{2}$$

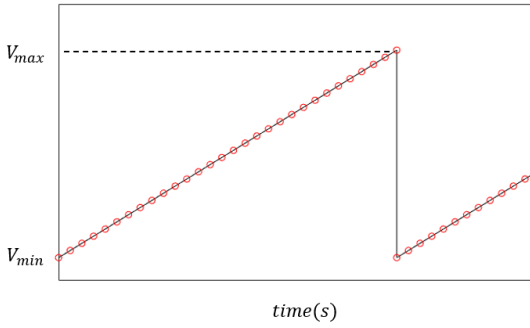


Fig. 5 Example of the digital sampling of an absolute encoder output signal

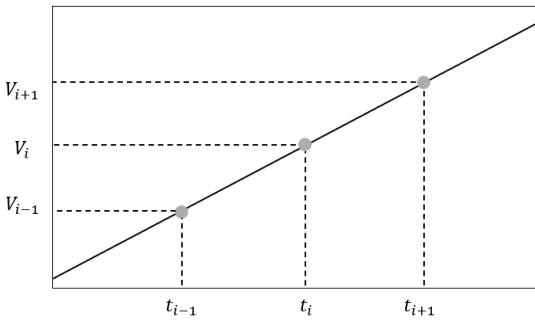


Fig. 6 Pulse output of absolute encoder installed at shaft with and without torsional vibration

Accordingly, the angular velocity variation of the shaft can be obtained by calculating angular velocity corresponding to each pulse in Fig. 4 using Eq. (2).

3.2 Rotational Vibration Measurement Using Absolute Encoder

In this section, the principle of the measurement of rotational vibration of the shaft using an ordinary absolute encoder will be explained. In the first step of the process, the analogue signal of is sampled with a specific sampling rate to obtain a series of digital signal as shown in Fig. 5.

In the case of the constant angular velocity, the slopes of individual digital data will be constant since the slopes are proportional to the angular velocity at the time slot. On the other hand, the slopes for the individual points will have variation with time when the shaft have rotational or tor-

sional vibration. So, the rotational vibration of the shaft can be calculated based on the slopes of individual data points in Fig. 6.

Figure 6 explains arbitrary 3 points on the sampled voltage data. Rotation angle (θ_i) and angular displacement ($\Delta\theta_i$) for the i^{th} time slot can be obtained as Eq. (3a) and Eq. (3b) following equations.

$$\theta_i = \frac{\theta_{\max}}{V_{\max}} \tag{3a}$$

$$\Delta\theta_i = (V_{i+1} - V_i) \frac{\theta_{\max}}{V_{\max}} \tag{3b}$$

Angular velocity of the i^{th} time slot can be obtained by dividing $\Delta\theta_i$ with time interval ($\Delta t_i = t_{i+1} - t_i$). In addition, since the voltage output of the encoder is sampled at a constant time interval, Δt_i is constant for all the data points as Δt . Consequently, angular velocities for each point can be calculated as Eq. (4).

$$\omega_i = \frac{\Delta\theta_i}{\Delta t} \tag{4}$$

By repeating this process for a specific time interval, the rotational velocity variations or rotational vibration of the shaft for the time interval can be obtained.

4. Validations of Rotational Vibration Measurement Process

In this section, two methods for the measurement of rotational vibration using incremental and absolute encoders will be compared numerically and experimentally. In the numerical comparison, the results from the numerical calculations simulating the measurements using incremental and absolute encoders will be compared with each other. Also, in the experimental comparison, rotational vibrations of the shaft measured with the two processes will be compared with each other.

4.1 Simulation

In this section, rotational vibration measurement

Table 1 Parameters of the rotating shaft for the simulation to verify measurement process

Description		Unit	Value
Shaft Rotation	Ω	r/min	150
	Ω_1	r/min	0.75
	Ω_2	r/min	1.5
	ϕ_1, ϕ_2	rad	0

Table 2 Parameters of the encoders and data recorders in the simulation to verify measurement process

Description		Unit	Value	
Encoder	Absolute	V_{max}	V	5.0
		θ_{max}	rad	32π
	Incremental	V_0	V	5.0
		PPR	-	100
Data Recorder	Absolute	f_s	Hz	$1e3$
		T	s	32
	Incremental	f_s	Hz	$1e5$
		T	s	10

results are simulated using computer simulations. The rotational speed of the shaft in these simulations is assumed to be as Eq. (5) with the first and second order rotational vibration along with nominal speed of rotation.

$$\omega(t) = \Omega + \Omega_1 \sin(\Omega t + \phi_1) + \Omega_2 \sin(2\Omega t + \phi_2) \quad (5)$$

In this equation Ω is the nominal speed of shaft and Ω_1 and Ω_2 are the magnitudes of the first and second order rotational vibration of the shaft. The rotation angle of the shaft can be obtained as Eq. (6) by integrating the speed of shaft with respect to time.

$$\theta(t) = \Omega t - \frac{\Omega_1}{\Omega} \cos(\Omega t + \phi_1) - \frac{\Omega_2}{2\Omega} \cos(2\Omega t + \phi_2) \quad (6)$$

In this simulation, the imaginary shaft is assumed to be rotating with the angular velocity given in Eq. (5) along with the parameters given in Table 1.

As explained above, the incremental encoder generates a square pulse as θ increases in pre-determined amount $2\pi/PPR$ and the absolute encoder generates a triangular pulse as θ increases by 2π

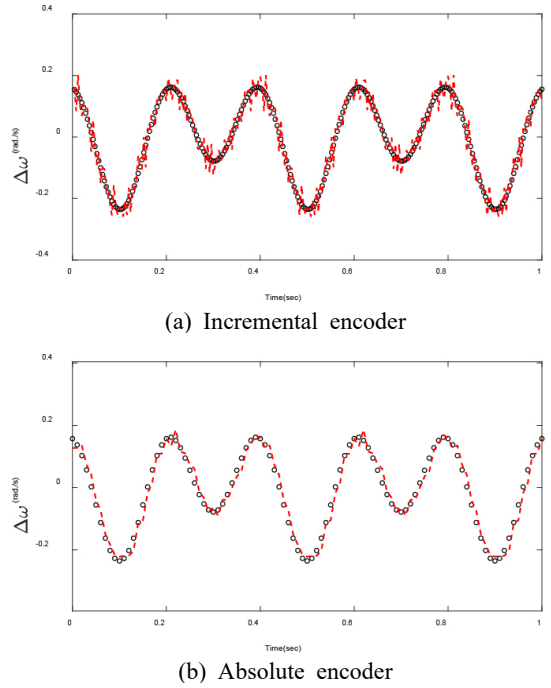


Fig. 7 Results of simulation to verify the measurement process (key: $\circ \circ \circ$ Theoretical, - - Simulation)

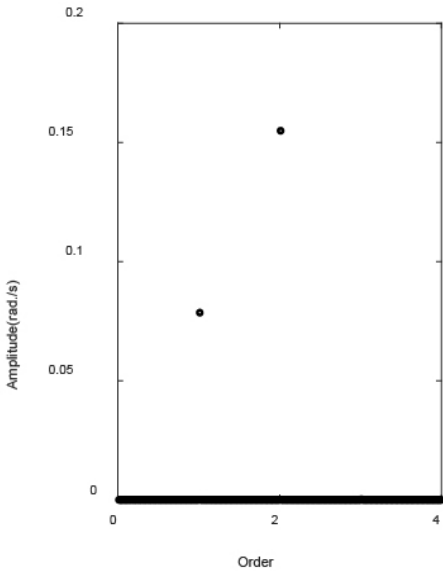
rad. Also, the output signals from the encoders are sampled by a data recorder for the analyses in the computers. The simulation parameters for the output signals and data recorders are given in Table 2.

In addition, 0.5% random error is added to the output signals from the encoders to consider the imperfectness of the encoders.

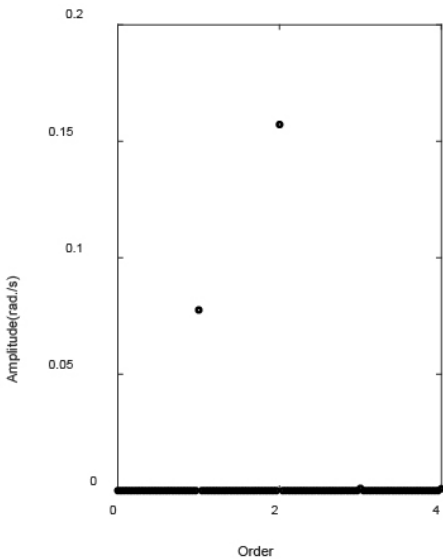
The simulation results for the two encoders under the conditions described in Table 1 and Table 2 are compared with each other and with rotational vibration which is introduced intentionally. The time data for the variations of angular velocities are compared in Fig. 7(a) and Fig. 7(b).

As one can see in the figures, $\Delta\omega_i$ from the incremental and absolute encoder and those for the intentionally introduced are quite well match each other. But, the effect of the random error is more dominant in the result of the absolute encoder. In addition, Fourier transform results of above data are explained in Fig. 8(a) and Fig. 8(b).

As shown in the figure, the first (2.5 Hz) and sec-



(a) Incremental encoder



(b) Absolute encoder

Fig. 8 Results of frequency analysis on the result of the simulation

ond order (5.0 Hz) rotational vibrations which are intentionally introduced in the simulations are clearly identified in the simulation results for the two processes. Also, the magnitudes of the vibration of individual components obtained from two processes are listed in Table 3 along with theoretical values.

As shown in the table, the magnitudes of the first

Table 3 The amplitude and errors of the first and second order rotational vibration from simulation

Quantity	Method	First order	Second order
Magnitude (rad/s)	Theory	0.0785	0.1571
	Incremental	0.0783	0.1553
	Absolute	0.0775	0.1572
Error (%)	Incremental	0.25	1.15
	Absolute	1.27	0.06

and second order vibrations are almost same as those of intentional rotational vibrations. Consequently, based on the results in Fig. 7 and Table 3, it can be concluded that the proposed processes have enough accuracies in the measurement of the rotational vibration of the shaft.

4.2 Experimental Verification

(1) Test set-up

The test setup for this study is explained in Fig. 9. As shown in the figure, the setup is composed of a shaft and a motor connected to each other with a universal joint. At the end of the shaft, an encoder is attached to measure rotational vibration of the shaft. The signals from the encoders are processed with a data recorder and a computer.

In the test with the incremental encoder, an incremental encoder W-60 from Autonics is used along with Virtex-II XC2V3000 FPGA data recorder. The output signal from the encoder sampled with the frequency of 1 MHz by the data recorder is analyzed by a computer to obtain the angular velocity of the shaft at the end of each pulse. For the experiment with the absolute encoder, an absolute encoder MCD-AC005-0412-M100-CAW from POSITAL Co. is used. This encoder generates a triangular pulse for every 16 rotations of the shaft.

If a pair of shafts are connected by a universal joint, the speeds of the input (ω_A) and output (ω_B) shafts have following relation⁽¹⁵⁾ of Eq. (7).

$$\frac{\omega_B}{\omega_A} = \frac{\cos \theta}{1 - \cos^2 \alpha \sin^2 \theta} \tag{7}$$

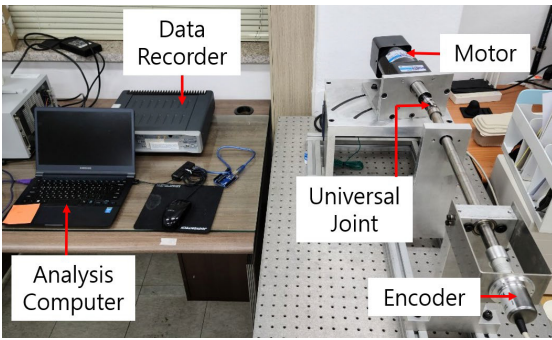


Fig. 9 The set-up for the experimental verification of the proposed process

In this equation, θ is the intersection angle of two shafts α is the rotation angle of the input shaft. As one can imagine from the equation, ω_B has two sinusoidal variation per revolution with max. speed of $\omega_A/\cos\theta$ and min. speed of $\omega_A\cos\theta$ ⁽¹⁵⁾. Obviously, there will be no dominant rotational vibration component when $\theta=0^\circ$ but there will be the second order rotational vibration with magnitude of 6% of nominal speed of the shaft when $\theta=20^\circ$.

(2) Test results

In this section, the rotational vibration of the shaft in the test setup measured with incremental and absolute encoders as explained in Section 2 and 3. The two results are compared with each other for the two cases $\theta=0^\circ$ and $\theta=20^\circ$. Also, the motor speed is set to 150 r/min.

The time signals from two methods are compared with each other in Fig. 10. As shown in the figure, the signals from two methods matches quite well each other in both cases.

In addition, in the case of $\theta=0^\circ$ the time signals have no dominant periodic variation while those for the case $\theta=20^\circ$ have apparent periodic variation with period of 0.2 s. In both cases, small irregular variations due to asymmetries, offset and unbalanced mass are included in the signals. Also, in the case of $\theta=20^\circ$ the amplitude of variation of period 0.2 s. is close to 1.0 rad/s. which is 6% of the nominal speed of the shaft. Also, the results from two processes with two types of encoders

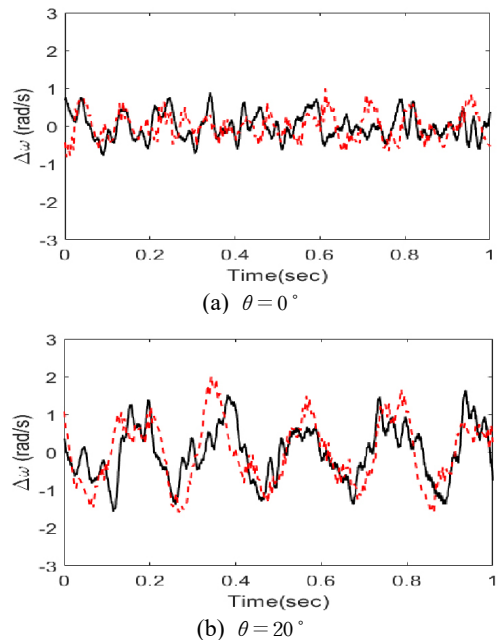


Fig. 10 Time data for the variation in the rotational speed of the shaft (key: — Absolute encoder, - - - Incremental encoder)

match quite well each other in two cases verifying that the proposed processes have sufficient accuracies in measuring rotational vibration.

As the next step, the frequency contents in rotational vibration of the shaft are identified by the frequency analyses in the form of order components of nominal speed of the shaft.

The results of frequency analyses of the outputs of processes with incremental and absolute encoders are compared in Fig. 11. In the case of $\theta=0^\circ$ no dominant order component can be identified in the results of both processes, as shown in the Fig. 11(a). But, in the case of $\theta=20^\circ$ the second order component is identified as a dominant component in the results of both processes, as shown in the Fig. 11(b). Also, the results from two processes are similar to each other, especially in the dominant order component. Also, both results include small components of vibration from uncertainties of the setup. Also, in the case of $\theta=20^\circ$ the amplitude of the second order vibration is close to 1.0 rad/s which is 6% of the nominal speed of the shaft.

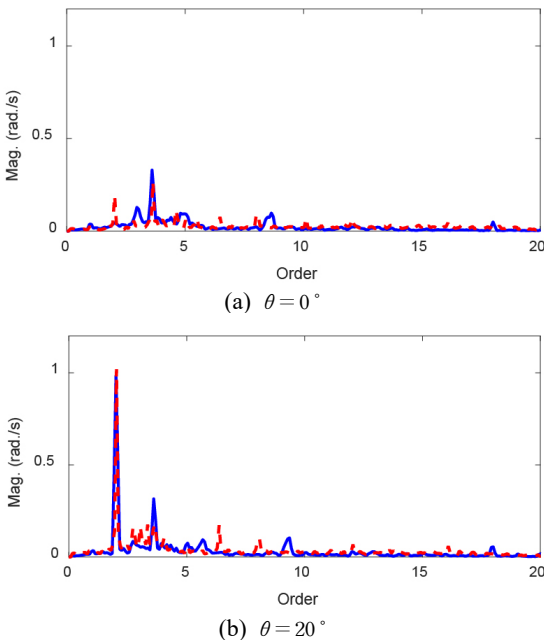


Fig. 11 Order contents in the rotational vibration of the shaft (key: — Absolute encoder, - - - Incremental encoder)

5. Conclusions

In this study, the measurement of rotational vibration with two types of encoders, i.e. incremental and absolute encoders are comparatively investigated. The measurement processes for two methods are introduced in the first stage. Then the proposed processes are examined using computer simulation verifying the validity of the proposed processes.

In the next step, the processes are experimentally validated using simple test setup. In the experiments, two types of encoders measured the rotational vibration of the shaft having the rotational vibration generated by a universal joint and a driving motor. The intersection angle of the universal joint θ is set to be 0° and 20° to control the rotational vibration in the shaft. The measurement results from two types of encoders are compared in time and frequency domains. The results from two types of encoders match quite well each other and theoretical calculations validating accuracies of the pro-

posed processes. When $\theta = 0^\circ$, no dominant component is observed but when $\theta = 20^\circ$ the second order vibration is identified as dominant component with the amplitude which is almost identical to the theoretical prediction. So, it can be concluded that the two procedures for the rotational vibration measurement using ordinary incremental and absolute encoders have sufficient accuracies.

The problems in the real applications of the proposed processes and corresponding countermeasures will be investigated in the subsequent studies.

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